



## Numerical Methods for FGM Composites Shells and Plates

Serçil Solmaz <sup>a\*</sup>, Ömer Civalek <sup>b</sup>

<sup>a,b</sup>Akdeniz University, Engineering Faculty, Civil Engineering Dept., Division of Mechanics

07058, Antalya-TURKIYE

E-mail address: [sercilsolmaz@hotmail.com](mailto:sercilsolmaz@hotmail.com) <sup>a\*</sup>, [civalek@yahoo.com](mailto:civalek@yahoo.com) <sup>b</sup>

ORCID numbers of authors:  
0000-0002-5410-971X <sup>a</sup>, 0000-0003-1907-9479 <sup>b</sup>

Received date: 14.04.2018

Accepted date: 25.05.2018

### Abstract

Main formulations for free vibration analysis of functionally graded composite shells have been given in numerical concept. Equations of motions for conical shells are listed in differential form. First-order shear deformation (FSDT) shell theory is used for obtaining the equations. Then two methods have been applied for solution. These methods are differential quadrature (DQ) and discrete singular convolution (DSC). The discrete forms of these equations have been given.

**Keywords:** Functionally graded composites, frequency, conical shells, annular plates, sector plates, DSC, HDQ.

### 1. Introduction

Functionally graded materials (FGM) are greatly used in different applications in engineering. Thus, many papers have been published for beams, plate and shell problems in order to obtain reasonable accurate results for design via different numerical methods [1-44]. By using the FSDT, the related governing equation for free vibration of conical shell can be written as

$$L_{11} + L_{12} + L_{13} + L_{14} + L_{15} - \rho h \cdot \omega^2 = 0 \quad (1)$$

$$L_{21} + L_{22} + L_{23} + L_{24} + L_{25} - \rho h \cdot \omega^2 = 0 \quad (2)$$

$$L_{31} \cdot U + L_{32} \cdot V + L_{33} \cdot W + L_{34} \cdot \Phi_x + L_{35} \cdot \Phi_y - \rho h \cdot \omega^2 = 0 \quad (3)$$

$$L_{41} \cdot U + L_{42} \cdot V + L_{43} \cdot W + L_{44} \cdot \Phi_x + L_{45} \cdot \Phi_y - \rho h^3 \cdot \omega^2 / 12 = 0 \quad (4)$$

$$L_{51} \cdot U + L_{52} \cdot V + L_{53} \cdot W + L_{54} \cdot \Phi_x + L_{55} \cdot \Phi_y - \rho h^3 \cdot \omega^2 / 12 = 0 \quad (5)$$



## 2. Solution by DSC method

By DSC method, governing differential equation of motion of truncated conical panel, Eqs. (1-5), can be discrete

$${}^{DSC}L_{11} \cdot U + {}^{DSC}L_{12} \cdot V + {}^{DSC}L_{13} \cdot W + {}^{DSC}L_{14} \cdot \Phi_x + {}^{DSC}L_{15} \cdot \Phi_y - \rho h \cdot \omega^2 = 0 \quad (6)$$

$${}^{DSC}L_{21} \cdot U + {}^{DSC}L_{22} \cdot V + {}^{DSC}L_{23} \cdot W + {}^{DSC}L_{24} \cdot \Phi_x + {}^{DSC}L_{25} \cdot \Phi_y - \rho h \cdot \omega^2 = 0 \quad (7)$$

$${}^{DSC}L_{31} \cdot U + {}^{DSC}L_{32} \cdot V + {}^{DSC}L_{33} \cdot W + {}^{DSC}L_{34} \cdot \Phi_x + {}^{DSC}L_{35} \cdot \Phi_y - \rho h \cdot \omega^2 = 0 \quad (8)$$

$${}^{DSC}L_{41} \cdot U + {}^{DSC}L_{42} \cdot V + {}^{DSC}L_{43} \cdot W + {}^{DSC}L_{44} \cdot \Phi_x + {}^{DSC}L_{45} \cdot \Phi_y - \rho h^3 \cdot \omega^2 / 12 = 0 \quad (9)$$

$${}^{DSC}L_{51} \cdot U + {}^{DSC}L_{52} \cdot V + {}^{DSC}L_{53} \cdot W + {}^{DSC}L_{54} \cdot \Phi_x + {}^{DSC}L_{55} \cdot \Phi_y - \rho h^3 \cdot \omega^2 / 12 = 0 \quad (10)$$

The coefficients of  $L_{ij}$  are:

$${}^{DSC}L_{11} = A_{11} \cdot \Xi_x^{(2)} + \frac{A_{11}}{R(x)} \sin \alpha \cdot \Xi_x^{(1)} - \frac{A_{22}}{R^2(x)} \cdot U(i) \cdot \sin^2 \alpha + \frac{A_{33}}{R^2(x)} \cdot \Xi_s^{(2)} \quad (11)$$

$${}^{DSC}L_{12} = \frac{(A_{12} + A_{33})}{R(x)} \sin \alpha \cdot \Xi_{xs}^{(2)} \frac{\partial^2 V}{\partial x \partial s} - \frac{(A_{22} + A_{33})}{R^2(x)} \sin \alpha \cdot \Xi_s^{(1)} \quad (12)$$

$${}^{DSC}L_{13} = \frac{A_{12}}{R(x)} \cos \alpha \cdot \Xi_x^{(1)} - \frac{A_{22}}{R^2(x)} \cdot W(i) \cdot \sin \alpha \cdot \cos \alpha \quad (13)$$

$${}^{DSC}L_{14} = B_{11} \cdot \Xi_x^{(2)} + \frac{B_{11}}{R(x)} \sin \alpha \cdot \Xi_x^{(1)} - \frac{B_{22}}{R^2(x)} \cdot \Psi_x(i) \sin^2 \alpha + \frac{B_{33}}{R^2(x)} \cdot \Xi_s^{(2)} \quad (14)$$

$${}^{DSC}L_{15} = \frac{(B_{12} + B_{33})}{R(x)} \cdot \Xi_x^{(1)} \cdot \Xi_s^{(1)} - \frac{(B_{22} + B_{33})}{R^2(x)} \cdot \Xi_s^{(1)} \cdot \sin \alpha \quad (15)$$

$${}^{DSC}L_{21} = \frac{(A_{12} + A_{33})}{R(x)} \cdot \Xi_x^{(1)} \cdot \Xi_s^{(1)} + \frac{(A_{22} + A_{33})}{R^2(x)} \sin \alpha \cdot \Xi_s^{(1)} \quad (16)$$

$${}^{DSC}L_{22} = A_{33} \Xi_x^{(2)} + A_{33} \frac{\sin \alpha}{R(x)} \Xi_s^{(1)}$$

$$- \frac{A_{33}}{R^2(x)} \cdot V(i) \cdot \sin^2 \alpha + \frac{A_{22}}{R^2(x)} \Xi_s^{(2)} - \frac{A_{44}}{R^2(x)} \cdot V(i) \cdot \cos^2 \alpha \quad (17)$$

$${}^{DSC}L_{23} = \frac{(A_{22} + A_{44})}{R^2(x)} \cdot \cos \alpha \cdot \Xi_s^{(1)} \quad (18)$$

$$^{DSC}L_{24} = \frac{(B_{12} + B_{33})}{R(x)} \cdot \Xi_x^{(1)} \cdot \Xi_s^{(1)} + \frac{(B_{22} + B_{33})}{R^2(x)} \sin \alpha \cdot \Xi_s^{(1)} \quad (19)$$

$$\begin{aligned} ^{DSC}L_{25} &= B_{33} \cdot \Xi_x^{(2)} + B_{33} \frac{\sin \alpha}{R(x)} \cdot \Xi_x^{(1)} \\ &- \frac{B_{33}}{R^2(x)} \cdot \Psi_s(i) \cdot \sin^2 \alpha + \frac{B_{22}}{R^2(x)} \cdot \Xi_s^{(2)} + A_{44} \cdot \frac{\cos \alpha}{R(x)} \cdot \Psi_s(i) \end{aligned} \quad (20)$$

$$^{DSC}L_{31} = -\frac{A_{12}}{R(x)} \cos \alpha \cdot \Xi_x^{(1)} - \frac{A_{22}}{R^2(x)} \cdot U(i) \cdot \sin \alpha \cdot \cos \alpha \quad (21)$$

$$^{DSC}L_{32} = -\frac{(A_{22} + A_{44})}{R^2(x)} \cos \alpha \cdot \Xi_s^{(1)} \quad (22)$$

$$^{DSC}L_{33} = A_{55} \cdot \Xi_x^{(2)} + \frac{A_{55}}{R(x)} \sin \alpha \cdot \Xi_s^{(1)} + \frac{A_{44}}{R^2(x)} \cdot \Xi_s^{(2)} - \frac{A_{22}}{R^2(x)} \cdot W(i) \cdot \cos^2 \alpha \quad (23)$$

$$\begin{aligned} ^{DSC}L_{34} &= A_{55} \cdot \Xi_x^{(1)} - \frac{B_{12}}{R(x)} \cos \alpha \cdot \Xi_x^{(1)} \\ &+ \frac{A_{55}}{R(x)} \cdot \Psi_x(i) \cdot \sin \alpha - \frac{B_{22}}{R^2(x)} \cdot \Psi_x(i) \cdot \sin \alpha \cdot \cos \alpha \end{aligned} \quad (24)$$

$$^{DSC}L_{35} = \frac{A_{44}}{R(x)} \cdot \Xi_s^{(1)} - \frac{B_{22}}{R^2(x)} \cdot \cos \alpha \cdot \Xi_s^{(1)} \quad (25)$$

$$^{DSC}L_{41} = B_{11} \cdot \Xi_x^{(2)} + \frac{B_{11}}{R(x)} \sin \alpha \cdot \Xi_x^{(1)} - \frac{B_{22}}{R^2(x)} \cdot U(i) \cdot \sin^2 \alpha + \frac{B_{33}}{R^2(x)} \cdot \Xi_s^{(2)} \quad (26)$$

$$^{DSC}L_{42} = \frac{(B_{12} + B_{33})}{R(x)} \cdot \Xi_x^{(1)} \cdot \Xi_s^{(1)} - \frac{(B_{22} + B_{33})}{R^2(x)} \sin \alpha \cdot \Xi_s^{(1)} \quad (27)$$

$$^{DSC}L_{43} = -A_{55} \cdot \Xi_x^{(1)} + B_{12} \frac{\cos \alpha}{R(x)} \cdot \Xi_x^{(1)} - \frac{B_{22}}{R^2(x)} \cdot W(i) \cdot \sin \alpha \cos \alpha \quad (28)$$

$$^{DSC}L_{44} = D_{11} \cdot \Xi_x^{(2)} + D_{11} \frac{\sin \alpha}{R(x)} \cdot \Xi_x^{(1)} - \frac{D_{22}}{R^2(x)} \sin^2 \alpha + \frac{D_{33}}{R^2(x)} \cdot \Xi_s^{(2)} - A_{55} \cdot \Psi_x(i) \quad (29)$$

$$^{DSC}L_{45} = \frac{(D_{12} + D_{33})}{R(x)} \cdot \Xi_x^{(1)} \cdot \Xi_s^{(1)} - \frac{(D_{22} + D_{33})}{R^2(x)} \cdot \Xi_s^{(1)} \sin \alpha \quad (30)$$

$$^{DSC}L_{51} = \frac{(B_{12} + B_{33})}{R(x)} \cdot \Xi_x^{(1)} \cdot \Xi_s^{(1)} + \frac{(B_{22} + B_{33})}{R^2(x)} \cdot \Xi_s^{(1)} \sin \alpha \quad (31)$$

$$\begin{aligned} {}^{DSC}L_{52} &= B_{33} \cdot \Xi_x^{(2)} + B_{33} \frac{\sin \alpha}{R(x)} \cdot \Xi_x^{(1)} \\ &- B_{33} \cdot \frac{\sin^2 \alpha}{R^2(x)} \cdot V(i) + \frac{B_{22}}{R^2(x)} \cdot \Xi_s^{(2)} + \frac{A_{44}}{R(x)} \cdot V(i) \cdot \cos \alpha \end{aligned} \quad (32)$$

$${}^{DSC}L_{53} = -\frac{A_{44}}{R(x)} \cdot \Xi_s^{(1)} + \frac{B_{22}}{R^2(x)} \cos \alpha \cdot \Xi_s^{(1)} \quad (33)$$

$${}^{DSC}L_{54} = \frac{(D_{12} + D_{33})}{R(x)} \cdot \Xi_x^{(1)} \cdot \Xi_s^{(1)} + \frac{(D_{22} + D_{33})}{R^2(x)} \sin \alpha \cdot \Xi_s^{(1)} \quad (34)$$

$$\begin{aligned} {}^{DSC}L_{55} &= D_{33} \cdot \Xi_x^{(1)} + D_{33} \frac{\sin \alpha}{R(x)} \cdot \Xi_x^{(1)} \\ &- \frac{D_{33}}{R^2(x)} \cdot \Psi_s(i) \cdot \sin^2 \alpha + \frac{D_{22}}{R^2(x)} \cdot \Xi_s^{(2)} - A_{44} \cdot \Psi_s(i) \end{aligned} \quad (35)$$

DSC derivation is given as

$$\Xi_x^n(\cdot) = \frac{\partial^{(n)}}{\partial x^{(n)}} = \sum_{k=-M}^M \delta_{\Delta, \sigma}^{(n)}(k \cdot \Delta x)(\cdot)_{i+k, j} \quad (36)$$

$$\Xi_s^n(\cdot) = \frac{\partial^{(n)}}{\partial s^{(n)}} = \sum_{k=-M}^M \delta_{\Delta, \sigma}^{(n)}(k \cdot \Delta s)(\cdot)_{i, j+k} \quad (37)$$

$$\Xi_x^1 \Xi_s^{(n-1)}(\cdot) = \frac{\partial^{(n)}}{\partial x \cdot \partial s^{(n-1)}} = \sum_{k=-M}^M \delta_{\Delta, \sigma}^{(1)}(k \cdot \Delta x)(\cdot)_{i+k, j} \sum_{k=-M}^M \delta_{\Delta, \sigma}^{(n-1)}(k \cdot \Delta s)(\cdot)_{i, k+j} \quad (38)$$

$$\Xi_x^{(n-1)} \Xi_s^1(\cdot) = \frac{\partial^{(n)}}{\partial x^{(n-1)} \partial s} = \sum_{k=-M}^M \delta_{\Delta, \sigma}^{(n-1)}(k \cdot \Delta x)(\cdot)_{i+k, j} \sum_{k=-M}^M \delta_{\Delta, \sigma}^{(1)}(k \cdot \Delta s)(\cdot)_{i, k+j} \quad (39)$$

### 3. Solution by DQ method

If DQ used above derivation can be define as

$$\Xi_x^n(*) = \frac{\partial^{(n)}}{\partial x^{(n)}}(*) = \sum_{k=1}^N C_{i+k, j}^{(n)}(i)(*)_{i+k, j} \quad (40)$$

$$\Xi_s^n(*) = \frac{\partial^{(n)}}{\partial s^{(n)}}(*) = \sum_{k=1}^N C_{i, j+k}^{(n)}(j)(*)_{i, j+k} \quad (41)$$

$$\Xi_x^1 \Xi_s^{(n-1)} (*) = \frac{\partial^{(n)}(*)}{\partial x \cdot \partial s^{(n-1)}} = \sum_{k=1}^N C_{i+k,j}(i) \sum_{k=1}^N C_{i,k+j}^{(n-1)}(j) (*)_{i,k+j} \quad (42)$$

$$\Xi_x^{(n-1)} \Xi_s^1 (*) = \frac{\partial^{(n)}(*)}{\partial x^{(n-1)} \partial s} = \sum_{k=1}^N C_{i,k+j}^{(n-1)}(j) \sum_{k=1}^N C_{i+k,j}^{(1)}(i) (*)_{i,k+j} \quad (43)$$

$C_{ijk}$  are weighting coefficients. The equations of motion are:

$${}^{DQ}L_{11} \cdot U + {}^{DQ}L_{12} \cdot V + {}^{DQ}L_{13} \cdot W + {}^{DQ}L_{14} \cdot \Phi_x + {}^{DQ}L_{15} \cdot \Phi_y - \rho h \cdot \omega^2 = 0 \quad (44)$$

$${}^{DQ}L_{21} \cdot U + {}^{DQ}L_{22} \cdot V + {}^{DQ}L_{23} \cdot W + {}^{DQ}L_{24} \cdot \Phi_x + {}^{DQ}L_{25} \cdot \Phi_y - \rho h \cdot \omega^2 = 0 \quad (45)$$

$${}^{DQ}L_{31} \cdot U + {}^{DQ}L_{32} \cdot V + {}^{DQ}L_{33} \cdot W + {}^{DQ}L_{34} \cdot \Phi_x + {}^{DQ}L_{35} \cdot \Phi_y - \rho h \cdot \omega^2 = 0 \quad (46)$$

$${}^{DQ}L_{41} \cdot U + {}^{DQ}L_{42} \cdot V + {}^{DQ}L_{43} \cdot W + {}^{DQ}L_{44} \cdot \Phi_x + {}^{DQ}L_{45} \cdot \Phi_y - \rho h^3 \cdot \omega^2 / 12 = 0 \quad (47)$$

$${}^{DQ}L_{51} \cdot U + {}^{DQ}L_{52} \cdot V + {}^{DQ}L_{53} \cdot W + {}^{DQ}L_{54} \cdot \Phi_x + {}^{DQ}L_{55} \cdot \Phi_y - \rho h^3 \cdot \omega^2 / 12 = 0 \quad (48)$$

In FGM material some properties are not constant:

$$E(z) = (E_c - E_m)V_c + E_m \quad (49)$$

$$\nu(z) = (\nu_c - \nu_m)V_c + \nu_m \quad (50)$$

For example if four-parameter power law is used then volume fractions are given for two cases.

$$\text{Case-1: } V_f = \left[ 1 - a \left( \frac{z}{h} + \frac{1}{2} \right) + b \left( \frac{z}{h} + \frac{1}{2} \right)^c \right]^p \quad (51)$$

$$\text{Case-2: } V_f = \left[ 1 - a \left( -\frac{z}{h} + \frac{1}{2} \right) + b \left( -\frac{z}{h} + \frac{1}{2} \right)^c \right]^p \quad (52)$$

#### 4. Conclusion

These equations can also be used for circular cylindrical shell and panel, annular, circular plates, sector and annular sector plates. Each methods have own advantages. But for higher modes, the method of DSC is more effective.

#### References

- [1] Reddy, J.N., Mechanics of Laminated Composite Plates and Shells: Theory and Analysis, New York: CRC Press; 2<sup>nd</sup> edition, 2003.

- [2] Qatu, M., *Vibration of Laminated Shells and Plates*, Academic Press, U.K., 2004.
- [3] Soedel, W., *Vibrations of Shells and Plates*, CRC Press; 3<sup>rd</sup> edition, 2004.
- [4] Leissa, A.W., *Vibration of Shells*, Acoustical Society of America, 1993.
- [5] Shen, H.S., *Functionally Graded Materials: Nonlinear Analysis of Plates and Shells*, CRC Press, 2009.
- [6] Elishakoff, I., Pentaras D., Gentilini C., Mechanics of Functionally Graded Material Structures, *World Scientific Publishing Conference*, 2015.
- [7] Ye, J., *Laminated composite plates and shells: 3D modeling*, Springer, 2003.
- [8] Timoshenko, S. and Woinowsky-Krieger, S., *Theory of Plates and Shells*, McGraw-Hill, New York; 2<sup>nd</sup> edition, 1959.
- [9] Liew, K.M., Zhao, X., Ferreira, A.J.M., A review of meshless methods for laminated and functionally graded plates and shells. *Compos Struct*, 93, 2031-2041, 2011.
- [10] Civalek, O., *Finite Element Analysis of Plates and Shell*, Firat University, Elazığ, 1988 (in Turkish).
- [11] Civalek, O. Geometrically non-linear static and dynamic analysis of plates and shells resting on elastic foundation by the method of polynomial differential quadrature (PDQ) [Ph. D. thesis]. Firat University, Elazığ, 2004 (in Turkish).
- [12] Qatu, M.S., Sullivan, R.W., Wang, W., Recent research advances on the dynamic analysis of composite shells: 2000–2009 Review Article. *Compos Struct*, 93, 14-31, 2010.
- [13] Ferreira A.J.M., Viola, E., Tornabene, F., Fantuzzi, N., Zenkour, A.M., Analysis of sandwich plates by generalized differential quadrature method. *Math Probl Eng*, 2013, 1-12, 2013.
- [14] Tornabene, F., Viola, E., Inman, D.J., 2-D differential quadrature solution for vibration analysis of functionally graded conical, cylindrical shell and annular plate structures. *J Sound Vib*, 328, 259-290, 2009.
- [15] Fantuzzi, N., Tornabene, F., Bacciocchi, M., Dimitri, R., Free vibration analysis of arbitrarily shaped Functionally Graded Carbon Nanotube-reinforced plates. *Compos Part B: Eng*, 115, 384-408, 2017.
- [16] Civalek, O., Vibration analysis of laminated composite conical shells by the method of discrete singular convolution based on the shear deformation theory. *Compos Part B Eng*, 45(1), 1001-1009 , 2013.
- [17] Civalek, O., The determination of frequencies of laminated conical shells via the discrete singular convolution method. *J Mech Mater Struct*; 1(1), 163-182, 2006.
- [18] Jin, G.Y., Su, Z., Shi, S., Ye, T.G., Gao, S.Y., Three-dimensional exact solution for the free vibration of arbitrarily thick functionally graded rectangular plates with general boundary conditions. *Compos Struct*, 108, 565-577, 2014.
- [19] Thai, H.-T., Kim, S.-E., A review of theories for the modeling and analysis of functionally graded plates and shells, *Compos Struct*, 128, 70-86, 2015.
- [20] Lei, Z.X., Liew, K.M., Yu, J.L., Free vibration analysis of functionally graded carbon nanotube-reinforced composite plates using the element-free kp-Ritz method in thermal environment. *Compos Struct*, 106, 128-138, 2013.
- [21] Shen, H.S., Zhang C.L., Thermal buckling and postbuckling behavior of functionally graded carbon nanotube-reinforced composite plates. *Mater Des*, 31, 3403–3411, 2010.
- [22] Ansari, R., Torabi, J., Faghih, M.S., Vibrational analysis of functionally graded carbon nanotube-reinforced composite spherical shells resting on elastic foundation using the variational differential quadrature method. *Eur J Mech-A Solid*, 60, 166–182, 2016.
- [23] Demir, Ç., Mercan, K., Civalek, O., Determination of critical buckling loads of isotropic, FGM and laminated truncated conical panel. *Compos Part B: Eng*, 94, 1-10, 2016.

- [24] Mercan, K., Civalek, O., DSC method for buckling analysis of boron nitride nanotube (BNNT) surrounded by an elastic matrix. *Compos Struct*, 143, 300-309, 2016.
- [25] Akgöz, B., Civalek, O., Buckling analysis of cantilever carbon nanotubes using the strain gradient elasticity and modified couple stress theories. *J Comp Theory Nanosci*, 8(9), 1821-1827, 2011.
- [26] Shao, Z., Shen, Z., He, Q., Wei, G.W., A generalized higher order finite-difference time-domain method and its application in guided-wave problems, *IEEE Transact Microwave Theory Tech*, 51, 856-861, 2003.
- [27] Civalek, O., Nonlinear dynamic response of MDOF systems by the method of harmonic differential quadrature (HDQ), *Struct Eng Mech*, 25 (2), 201-217, 2007.
- [28] Bao, W., Sun, F., Wei, G.W., Numerical methods for the generalized Zakharov system, *J Comput Physics*, 190, 201–228, 2003.
- [29] Civalek, O., Nonlinear dynamic response of laminated plates resting on nonlinear elastic foundations by the discrete singular convolution-differential quadrature coupled approaches, *Compos Part B Eng*, 50, 171-179, 2013.
- [30] Civalek, O., Korkmaz, A., Demir, Ç., Discrete singular convolution approach for buckling analysis of rectangular Kirchhoff plates subjected to compressive loads on two opposite edges. *Adv Eng Softw*, 41, 557-560, 2010.
- [31] Civalek, O., Analysis of thick rectangular plates with symmetric cross-ply laminates based on first-order shear deformation theory. *J Compos Mater*, 42, 2853–2867, 2008.
- [32] Wang, X., Wang, Y., Xu, S., DSC analysis of a simply supported anisotropic rectangular plate. *Compos Struct*, 94, 2576-2584, 2012.
- [33] Baltacıoğlu, A.K., Civalek, Ö., Akgöz, B., Demir, F., Large deflection analysis of laminated composite plates resting on nonlinear elastic foundations by the method of discrete singular convolution. *Int J Pres Ves Pip*, 88, 290-300, 2011.
- [34] Civalek, O., Akgöz, B., Vibration analysis of micro-scaled sector shaped graphene surrounded by an elastic matrix. *Comp Mater Sci*, 77, 295-303, 2013.
- [35] Gürses, M., Civalek, O., Korkmaz, A., Ersoy, H., Free vibration analysis of symmetric laminated skew plates by discrete singular convolution technique based on first-order shear deformation theory. *Int J Numer Methods Eng*, 79, 290-313, 2009.
- [36] Baltacıoğlu, A.K., Akgöz, B., Civalek, O., Nonlinear static response of laminated composite plates by discrete singular convolution method. *Compos Struct*, 93, 153-161, 2010.
- [37] Gürses, M., Akgöz, B., Civalek, O., Mathematical modeling of vibration problem of nano-sized annular sector plates using the nonlocal continuum theory via eight-node discrete singular convolution transformation. *Appl Math Comput*, 219, 3226–3240, 2012.
- [38] Civalek, O., Mercan, K., Demir, C., Vibration analysis of FG cylindrical shells with power-law index using discrete singular convolution technique. *Curved and Layer Struct*, 3, 82-90, 2016.
- [39] Tong, L., Free vibration of laminated conical shells including transverse shear deformation. *Int J Solids Struct*, 31, 443–456, 1994.
- [40] Wang, Q., Shi, D., Liang, Q., Shi, X., A unified solution for vibration analysis of functionally graded circular, annular and sector plates with general boundary conditions. *Compos Part B*, 88, 264-294, 2016.
- [41] Mercan, K., Civalek, O., Buckling analysis of silicon carbide nanotubes (SiCNTs). *Int J Eng Appl Sci*, 8 (2), 101-108, 2016.
- [42] Civalek, O., Çatal, H.H., Plakların diferansiyel quadrature metodu ile stabilité ve titreşim analizi. *Teknik Dergi*, 14 (1), 2835-2852, 2003.
- [43] Civalek, O., Diferansiyel quadrature metodu ile elastik çubukların statik dinamik ve burkulma analizi, *XVI Mühendislik Teknik Kongresi*, ODTU, Ankara, Kasım 2001.

- [44] Civalek, O., Numerical solutions to the free vibration problem of Mindlin sector plates using the discrete singular convolution method. *Int J Struct Stab Dyn*, 9 (2), 267-284, 2009.